Amendments to the Specification:

Before paragraph [0001] add the following new sub-headings and paragraph:

-- PRIORITY CLAIM

This is a U.S. national stage of application No. PCT/EP2003/007207, filed on 05 July 2003.

BACKGROUND OF THE INVENTION

1. Field of the Invention --

Amend paragraph [0001] as follows:

[0001]` The invention pertains to an axial friction bearing for supporting the rotating shaft of an exhaust gas turbocharger connected to a lubricating oil circuit ~~according to the introductory clause of Claim 1~~.

Before paragraph [0002] add the following new sub-heading:

-- 2. Description of the Related Art --

Amend paragraph [0004] as follows:

[0004] For example, U.S. Patent No. 6,024,495 discloses an axial friction bearing of this type ~~is known from EP 0 840 027 A2~~, which ~~consists of~~ includes a bearing body, permanently connected to a bearing housing; a bearing comb, which rotates along with the shaft; and at least one lubricating gap, provided between the bearing box and the bearing comb. The gap is formed between the profiled surface of a ring and a flat ~~slide~~ sliding surface and is connected to an oil supply. A shoulder of the shaft, that is, the bearing comb seated on

the shaft, therefore runs at least indirectly against an end surface of the stationary bearing housing. The surface of the ring has several radially oriented longitudinal oil grooves.

Amend paragraph [0007] as follows:

[0007] For example, according to ~~EP 0 840 027 A2~~ U.S. Patent No. 6,024,495, axial friction bearings of this type are provided with a convergent lubricating gap, which is advantageous for developing hydrodynamic pressure, by machining wedge surfaces oriented in the circumferential direction into bearings in such a way that the lubricating gap is smallest in the area of the adjacent lubricating oil groove.

Amend paragraph [0012] as follows:

[0012] An axial bearing with a free-floating disk, which rotates around or with the shaft, is known especially from ~~EP 0 840 027 A2~~ U.S. Patent No. 6,024,495, which is incorporated herein by reference. Here the wedge surface is bordered radially by an outer sealing web, and the sealing web is interrupted by a radially outward-leading dirt and/or cooling groove. The goal of this sealing web, which borders both the lubricating oil grooves and the wedge surfaces, is to decrease the amount of lubricating oil that can escape radially to the outside in the area of the wedge surfaces. This solution, too, suffers from the disadvantage that a design of this type also tends to fail spontaneously (see below).

Before paragraph [0014] add the following new sub-heading:

-- **SUMMARY OF THE INVENTION** --

Amend paragraph [0015] as follows:

[0015] This is achieved by a profiled annular bearing surface having a plurality of radially extending grooves which are open on the outside circumference, a plurality of coplanar flat trap surfaces located between respective pairs of adjacent said grooves, and a plurality of wedge surfaces located between adjacent pairs of respective grooves, wherein each wedge surface forms a lubricating oil gap which narrows circumferentially toward an adjacent flat trap surface and which narrows radially toward the outside circumference ~~the characterizing features of Claim 1 in conjunction with the features of the introductory clause~~.

Delete paragraph [0021].

Before paragraph [0022] add the following new sub-heading:

-- **BRIEF DESCRIPTION OF THE DRAWINGS** --

Amend paragraph [0023] as follows:

[0023] ~~Figure 2 shows~~ Figures 2A-2C show three cross sections through the ring-shaped bearing surface of Figure 1 proceeding in the radial direction from the inside out, namely, along lines A-A, B-B, and C-C in Figure 1.

Before paragraph [0024] add the following new sub-heading:

-- **DETAILED DESCRIPTION OF THE PRESENTLY PREFERRED EMBODIMENTS** --

Amend paragraph [0026] as follows:

[0026] An axial friction bearing part has a bearing surface 3 in the form of a profiled ~~ring~~ annular surface, which has several lubricating oil grooves 2 distributed at regular intervals around the circumference of the ~~ring-shaped~~ annular surface, proceeding radially across that surface. Between each pair of lubricating oil grooves is a wedge surface 1 and an adjacent flat trap surface 5.

Amend paragraph [0029] as follows:

[0029] The convergent orientation of the wedge surfaces 1 in the circumferential direction is known from~~, for example, EP 0 840 027 A2~~ U.S. Patent No. 6,024,495. The additional convergent orientation of these wedge surfaces 1 in the radial direction is illustrated in the present ~~Figure 2~~ Figures 2A-2C.

Amend paragraph [0030] as follows:

[0030] Thus the cross section A-A in the area of the inside circumference of the ring-shaped surface 3 close to the shaft shows that the thickness of the lubricating gap (i.e., the depth of wedge) is still comparatively large across the wedge surface. This thickness decreases, however, as we proceed from section B-B to section C-C toward the outside circumference of the ring-shaped surface 3 as far as the level of the trap surface 5. That is, the wedge surface ~~is oriented convergently~~ converges from the inside circumference ~~to~~ towards circumferential web 6 on the outside circumference. As shown in Figure 2C, the top surface of web 6 is at the level of trap surface 5, except where channels 4 interrupt the web 6. The

additional convergent orientation of the wedge surfaces in the circumferential direction of the ring surface 3 can be seen especially clearly in cross section A-A. From this (two-dimensional) superimposition of these two orientations, there results a helical wedge surface 1, which, in contrast to the embodiment of a "chambering" of the wedge surfaces toward the outside by means of a sealing web (see ~~EP 0 840 027 A2~~ U.S. Patent No. 6.024,495), allows a thoroughly sufficient amount of lubricating oil to flow away in the radial direction over each wedge surface 1. In the manner previously described, furthermore, this design of the wedge surface takes advantage of the radial component of the lubricating oil volume flow rate to build up additional pressure, whereas the previously known variant with a sealing web makes it possible for the pressure to build up only in the circumferential direction of the ring-shaped surface, leading to the problems discussed in the introduction.

Page 8, before claim 1, add the following new sub-heading:

-- What is claimed is: --